Relation of friction coefficient classes to guideline values for various materials/surfaces and types of lubrication, for screw connections

according to VDI 2230, edition 2015

The friction coefficients $\mu_{G}, \, \mu_{K}$ display variations since they are dependent on several factors, e.g. the material combinations, the quality of the surface finish (depth of roughness), the surface treatment (plain, blackened, galvanically zinc coated, zinc flake

coatings, etc.) and the method of lubrication (with/without oil, molybdenum disulfide, molycoat paste, anti-friction coating etc)! The following tables give friction coefficients for threads and for bearing surfaces.

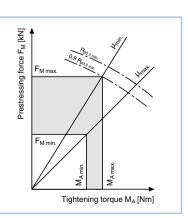
Friction coeff.	Range for	Typical examples for:	
class	μ_{G} and μ_{K}	Material/surfaces	Lubrication
A	0,04-0,10	metallic, bright-polished black tempered phosphated galvanized coatings such as Zn, Zn/Fe, Zn/Ni zinc laminated coatings	solid lubricants such as MoS ₂ , graphite, PTFE, PA, PE, PI in lubricating lacquers, or in pastes wax glazes, wax dispersions
В	0,08-0,16	metallic, bright-polished black tempered phosphated galvanized coatings such as Zn, Zn/Fe, Zn/Ni zinc laminated coatings Al and Mg alloys hot-dip galvanized organic coatings austenitic steel	solid lubricants such as MoS ₂ , graphite, PTFE, PA, PE, PI in lubricating lacquers, or in pastes, wax glazes, wax dispersions, greases, oils, as-delivered condition MoS ₂ , graphite, wax dispersions with integrated solid lubrication or wax dispersion solid lubricants or waxes;
С	0,14-0,24	austenitic steel	pastes wax dispersions, pastes
		metallic, bright-polished phosphated galvanic coatings such as Zn, Zn/Fe, Zn/Ni	as delivered state (lightly oiled)
		non electrolytically applied zinc adhesive	none
D	0,20-0,35	austenitic steel galvanic coatings such as Zn, Zn/Fe hot-dip galvanized	oil none
E	≥ 0,30	galvanised coatings such as Zn/Fe, Zn/Ni austenitic steel Al and Mg alloys	none

The data in the table is valid at room temperature.

The aim is to **achieve** coefficients of friction which fit into the **friction coefficient class B** in order to apply as high a preload as possible with low scatter. This does not automatically mean using the smallest values and that the friction coefficient scatter present corresponds to the class spread.

For a safe and secure mounting it is important to define the conditions for friction very precisely and to restrict their variations as much as possible.

If there is a large variation the desired preload force can vary considerably. In contrast to this the normal range of tolerance for the tightening torque has only a limited effect.



 μ_G = coefficient of friction in the thread

- μ_{K} = coefficient of friction in the head bearing area
- $\mu_{\text{T}}~$ = ~ coefficient of friction at the interface

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